# October 21 Math 2306 sec. 51 Fall 2024

#### **Section 11: Linear Mechanical Equations**

We were considering the displacement from equilibrium, x(t), of an object of mass *m* suspended from a flexible spring with *spring constant k*. In the absence of any sort of damping or external forces, the object exhibits **simple harmonic motion**.

The displacement is subject to the second order, linear, homogeneous differential equation

$$mx'' + kx = 0$$
 i.e.,  $x'' + \omega^2 x = 0$ ,

where the parameter

$$\omega^2 = \frac{k}{m}$$

# Equilibrium & Displacment in Equilibrium



Figure: Hooke's law states that the displacement in equilibrium,  $\delta x$ , is related to the object's weight, *W*, via  $W = k \delta x$ . We'll use the convention

x > 0 above equilibrium, and x < 0 below equilibrium. **Remark:** Since  $W = mg = k\delta x$ ,  $\omega^2 = \frac{k}{m} = \frac{g}{\delta x}$ .

#### Simple Harmonic Motion

With initial displacement  $x_0$  and initial velocity,  $x_1$ , the position x(t) at time *t* satisfies the IVP

$$x'' + \omega^2 x = 0$$
,  $x(0) = x_0$ ,  $x'(0) = x_1$ .

Characteristics of the system include

• the period 
$$T = \frac{2\pi}{\omega}$$
,

• the frequency 
$$f = \frac{1}{T} = \frac{\omega}{2\pi}^{1}$$

- the circular (or angular) frequency  $\omega$ , and
- the amplitude or maximum displacement  $A = \sqrt{x_0^2 + (x_1/\omega)^2}$

<sup>&</sup>lt;sup>1</sup>Various authors call *f* the natural frequency and others use this term for  $\omega$ .

#### Amplitude and Phase Shift

We can formulate the solution in terms of a single sine (or cosine) function. Letting

$$x(t) = x_0 \cos(\omega t) + \frac{x_1}{\omega} \sin(\omega t) = A \sin(\omega t + \phi) = A \cos(\omega t - \hat{\phi})$$

where the amplitude

$$A=\sqrt{x_0^2+(x_1/\omega)^2},$$

and the **phase shift**  $\phi$  or  $\hat{\phi}$  must be defined by

$$\sin \phi = \frac{x_0}{A}$$
, and  $\cos \phi = \frac{x_1}{\omega A}$  or  $\cos \hat{\phi} = \frac{x_0}{A}$ , and  $\sin \hat{\phi} = \frac{x_1}{\omega A}$ .  
Note,  $\phi + \hat{\phi} = \frac{\pi}{2}$  (up to an integer multiple of  $2\pi$ ).

# Example

A 4 pound weight stretches a spring 6 inches. The mass is released from a position 4 feet above equilibrium with an initial downward velocity of 24 ft/sec. Find the equation of motion in the form  $x = A\sin(\omega t + \phi)$ , and identify the period, amplitude, phase shift, and frequency of the motion. (Take g = 32 ft/sec<sup>2</sup>.)

From the weight W = 4lb and displacement in equilibrium  $\delta x = \frac{1}{2}$  ft, we found the mass and spring constant

$$m = \frac{1}{8}$$
 slugs, and  $k = 8 \frac{\text{lb}}{\text{ft}}$ .

This makes

$$\omega^2 = 64 \ \frac{1}{\sec^2}.$$

We ended up with the ODE

$$\frac{1}{8}x'' + 8x = 0 \implies x'' + 64x = 0$$

We need 
$$\chi(0) = d \chi'(0)$$
  
 $\chi(0) = 4 (4t; dbove equilibrium)$   
 $\chi'(0) = -24 (24 t) rec down word)$   
The Characteristic equation  $\Gamma^2 + 64 = 0$   
 $\Gamma^2 = -64 \Rightarrow \Gamma = \pm 8i$   
 $\chi(4) = C_1 G_3(84) + C_2 Sin(84)$   
 $\chi'(0) = -8C_1 Sin(84) + 8(-2 Cos(84))$   
 $\chi(0) = C_1 G_3(0) + C_2 Sin(0) = 4 \Rightarrow C_1 = 4$   
 $\chi'(0) = -8C_1 Sin(0) + 8C_2 Cos(0) = -24 \Rightarrow 8C_2 = -24$   
 $\chi'(0) = -8C_1 Sin(0) + 8C_2 Cos(0) = -24 \Rightarrow 8C_2 = -24$   
 $C_2 = -3$ 

$$X(t) = 4 \cos(\theta t) - 3 \sin(\theta t)$$
Note that amplitude  $A = \sqrt{4^2 + (-3)^2} = 5$ 
So  $X(t) = 5 \sin(\theta t + \phi)$  where
Sin  $\phi = \frac{4}{5}$  and  $\cos \phi = \frac{-3}{5}$ 
The period  $T = \frac{2\pi}{5} = \frac{2\pi}{8} = \frac{\pi}{4}$ 
Uncer frequency  $f = \frac{4}{7}$ 
The phase shift  $\phi = \cos^{-1}\left(\frac{-3}{5}\right)$ 
 $\phi \approx 2.21$ 

 $\phi \approx 127^{\circ}$ 

# Free Damped Motion



Figure: If a damping force is added, we'll assume that this force is proportional to the instantaneous velocity.

### Free Damped Motion

Now we wish to consider an added force corresponding to damping—friction, a dashpot, air resistance.

Total Force = Force of spring + Force of damping

$$m\frac{d^2x}{dt^2} = -b\frac{dx}{dt} - kx \implies \frac{d^2x}{dt^2} + 2\lambda\frac{dx}{dt} + \omega^2 x = 0$$
  
where  
$$2\lambda = \frac{b}{m} \text{ and } \omega = \sqrt{\frac{k}{m}}.$$

Three qualitatively different solutions can occur depending on the nature of the roots of the characteristic equation

$$r^2 + 2\lambda r + \omega^2 = 0$$
 with roots  $r_{1,2} = -\lambda \pm \sqrt{\lambda^2 - \omega^2}$ .



Figure: Two distinct real roots. No oscillations. Approach to equilibrium may be slow.



Figure: One real root. No oscillations. Fastest approach to equilibrium.



Figure: Complex conjugate roots. Oscillations occur as the system approaches (resting) equilibrium.

## Comparison of Damping



Figure: Comparison of motion for the three damping types.

# Example

A 2 kg mass is attached to a spring whose spring constant is 12 N/m. The surrounding medium offers a damping force numerically equal to 10 times the instantaneous velocity. Write the differential equation describing this system. Determine if the motion is underdamped, overdamped or critically damped.

The model is  

$$mx''+bx'+kx=0$$
  
[tere,  $m=2 kg$ ,  $k=12 N/m$ ,  $b=10$   
our oper is  
 $2x''+10x'+12 x = 0$ .

In standard firm, x'' + 5x' + 6x = 0

The chara deristic equ is

 $(^{2} + 5r + 6 = 0)$ 

which factors as (r+2)(r+3)=0

r=-2 or r=-3

1.1

we have two distinct real roots, so the system is over damped.

The ODE is 
$$\chi'' + S\chi' + 6\chi = 0$$
  
so  $Z\lambda = 5$ , and  $\omega^2 = 6$   
so  $\lambda = \frac{5}{2}$  and  
 $\lambda^2 - \omega^2 = (\frac{5}{2})^2 - 6 = \frac{35}{4} - \frac{34}{4} = \frac{1}{4} > 0$   
Again, the system is over damped.

Note, we didn't have to do this second test. But it shows that our conclusion based on the roots matches the condition related to the parameters, lambda and omega.