October 28 Math 2306 sec. 53 Fall 2024

Section 11: Linear Mechanical Equations

The basic model for the position, *x*(*t*), at time *t* of an object of mass *m* suspended from a flexible spring, possibly subjected to linear damping or external forcing is

$$
mx'' + bx' + kx = f(t),
$$

where *b* is the damping coefficient, *k* the spring constant, and *f* an external driving force.

In the absence of damping, $b = 0$. In the absence of driving, $f(t) = 0$.

Forced Undamped Motion and Resonance

If damping is negligible, and the system is driven by a simple oscillator, $f(t) = f_0 \cos(\gamma t)$ or $f(t) = f_0 \sin(\gamma t)$, the differential equation becomes

$$
x'' + \omega^2 x = F_0 \cos(\gamma t) \quad \text{or} \quad x'' + \omega^2 x = F_0 \sin(\gamma t),
$$

where $\omega^2 = \frac{k}{n}$ $\frac{k}{m}$ and $F_0 = \frac{f_0}{m}$.

> **Pure resonance** occurs when the forcing frequency γ matches the natural frequency ω .

$$
\gamma = \omega
$$

Even near resonance, $\gamma \approx \omega$, gives rise to large amplitude oscillations and can cause damaging vibrations.

Example

A 3 kg mass is attached to a spring with spring constant 48 N/m. Assume damping is negligible and the system is driven by a force $f(t) = F_0 \cos(\gamma t)$.

(a) What value of γ would induce pure resonance?

We found that
$$
\omega^2 = \frac{k}{m} = \frac{48}{3} = 16
$$
. So the resonance frequency $\gamma = 4$.
\n(b) If the object starts from rest from the equilibrium position, find the displacement in the pure resonance case.

The differential equation in the resonance case is

$$
3x'' + 48x = F_0 \cos(\omega t) \quad \Longrightarrow \quad x'' + 16x = \frac{F_0}{3} \cos(4t).
$$

The initial conditions are $x(0) = x'(0) = 0$.

$$
x'' + 16x = \frac{F_0}{3}\cos(4t), \ x(0) = 0, \ x'(0) = 0
$$

The complementary solution is

$$
x_c = c_1 \cos(4t) + c_2 \sin(4t).
$$

 55 $(25)^{16^{30}}$ $(25)^{16}$

We were finding *x^p* in the form

$$
x_p = At\cos(4t) + Bt\sin(4t).
$$

Note that

$$
\begin{array}{rcl}\n\text{(b)} & x_p & = & At\cos(4t) + Bt\sin(4t) \\
x'_p & = & A\cos(4t) + B\sin(4t) - 4At\sin(4t) + 4Bt\cos(4t) \\
\text{(c)} & x''_p & = & -8A\sin(4t) + 8B\cos(4t) - 16At\cos(4t) - 16Bt\sin(4t)\n\end{array}
$$

$$
x'' + 16x = \frac{F_0}{3} \cos(4t)
$$

\n
$$
t c_{0s}(4t) (-16A + 16A) + t 5 \cdot n(4t) (-16B + 16B)
$$

\n
$$
+ c_{0s}(4t) (-8B) + 5 \cdot n(4t) (-8A)
$$

\n
$$
= \frac{F_0}{3} c_{0s}(4t) + O \cdot 5 \cdot n(4t)
$$

\n
$$
\frac{F_0}{3} = 8B, O = -8A \Rightarrow A = 0, S = \frac{F_0}{24}
$$

\n
$$
x_0 = \frac{F_0}{24} t 5 \cdot n(4t) - 4
$$

\n
$$
x(t) = c_0 c_{0s} 4t + c_0 5 \cdot n 4t + \frac{F_0}{24} t 5 \cdot n 4t
$$

 $A_{PP}I_{3} \times (0.5 \times 10^{16}) = 0.5$

$$
x'(k) = -4c, s \cdot 4 + 4c, s \cdot 8 + \frac{F_0}{24} s \cdot 4 + \frac{F_0}{6} 6c_{os}4 +
$$

$$
X(6) = C_1C_6s0 + C_2s \cdot 6 + \frac{F_0}{24}(6)s \cdot 6 = 0
$$

$$
C_1 = 0
$$

$$
X'(6) = -4C_1s \cdot 6 + 4c_2s \cdot 6 + \frac{F_0}{24}s \cdot 6 + \frac{F_0}{6}(6)c_0s0 = 0
$$

$$
4(C_2 = 0) = 0
$$

$$
C_3 = 0
$$

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The position of the object is
\n
$$
x(t) = \frac{F_0}{24}6S_m 4t
$$