

Section 11: Linear Mechanical Equations

Simple Harmonic Motion

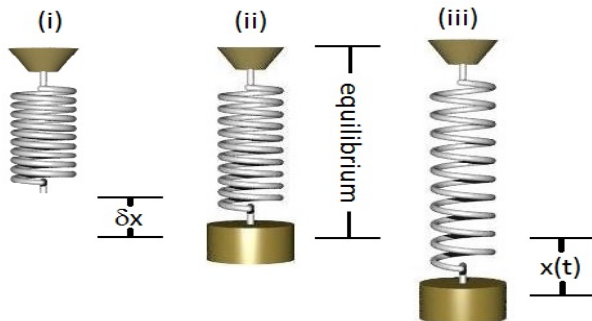


Figure: We track the displacement x from the equilibrium position. Recall δx is the displacement **in** equilibrium. We assume there are no damping nor external forces (for now).

Simple Harmonic Motion

We derive the equation for displacement. Given initial displacement x_0 and initial velocity x_1 ,

$$x'' + \omega^2 x = 0, \quad x(0) = x_0, \quad x'(0) = x_1, \quad (1)$$

where $\omega^2 = \frac{k}{m} = \frac{g}{\delta x}$. The solution is

$$x(t) = x_0 \cos(\omega t) + \frac{x_1}{\omega} \sin(\omega t) \quad (2)$$

called the **equation of motion**¹.

k -spring constant, m -object's mass, g -acceleration due to gravity.

¹ **Caution:** The phrase equation of motion is used differently by different authors.

Some use this phrase to refer the IVP (1). Others use it to refer to the **solution** to the IVP such as (2).

Simple Harmonic Motion

$$x(t) = x_0 \cos(\omega t) + \frac{x_1}{\omega} \sin(\omega t)$$

Characteristics of the system include

- ▶ the period $T = \frac{2\pi}{\omega}$,
- ▶ the frequency $f = \frac{1}{T} = \frac{\omega}{2\pi}$
- ▶ the circular (or angular) frequency ω , and
- ▶ the amplitude or maximum displacement $A = \sqrt{x_0^2 + (x_1/\omega)^2}$

²Various authors call f the natural frequency and others use this term for ω .

Amplitude and Phase Shift

We can formulate the solution in terms of a single sine (or cosine) function. Letting

$$x(t) = x_0 \cos(\omega t) + \frac{x_1}{\omega} \sin(\omega t) = A \sin(\omega t + \phi)$$

requires

$$A = \sqrt{x_0^2 + (x_1/\omega)^2},$$

and the **phase shift** ϕ must be defined by

$$\sin \phi = \frac{x_0}{A}, \quad \text{with} \quad \cos \phi = \frac{x_1}{\omega A}.$$

Amplitude and Phase Shift (alternative definition)

We can formulate the solution in terms of a single sine (or cosine) function. Letting

$$x(t) = x_0 \cos(\omega t) + \frac{x_1}{\omega} \sin(\omega t) = A \cos(\omega t - \hat{\phi})$$

requires

$$A = \sqrt{x_0^2 + (x_1/\omega)^2},$$

and this **phase shift** $\hat{\phi}$ must be defined by

$$\cos \hat{\phi} = \frac{x_0}{A}, \quad \text{with} \quad \sin \hat{\phi} = \frac{x_1}{\omega A}.$$

Example

An object stretches a spring 6 inches in equilibrium. Assuming no driving force and no damping, set up the differential equation describing this system.

No driving and no damping \Rightarrow
simple harmonic motion.

The ODE is $mx'' + kx = 0$,
in standard form

$$x'' + \omega^2 x = 0$$

we need to identify ω^2 .

We don't have enough info to find

m and k, but we can use

$$\omega^2 = \frac{g}{\delta x}.$$

The displacement in equilibrium $\delta x = 6 \text{ in.}$

In US units $g = 32 \text{ ft/sec}^2$

$$\delta x = 6 \text{ in} \left(\frac{1 \text{ ft}}{12 \text{ in}} \right) = \frac{1}{2} \text{ ft}$$

$$\omega^2 = \frac{32 \text{ ft/sec}^2}{\frac{1}{2} \text{ ft}} = 64 \frac{1}{\text{sec}^2}$$

The ODE is

$$x'' + 64x = 0$$

Example

A 4 pound weight stretches a spring 6 inches. The mass is released from a position 4 feet above equilibrium with an initial downward velocity of 24 ft/sec. Find the equation of motion, the period, amplitude, phase shift, and frequency of the motion. (Take $g = 32 \text{ ft/sec}^2$.)

The ODE is $mx'' + kx = 0$, or in standard form $x'' + \omega^2 x = 0$.

Since $\delta x = 6 \text{ in}$, we know $\omega^2 = 64$.

Let's find m and k .

$$\text{The mass } m = \frac{W}{g} = \frac{4 \text{ lb}}{32 \text{ ft/sec}^2} = \frac{1}{8} \text{ slug}$$

The spring constant $k = \frac{W}{\delta x} = \frac{41b}{\frac{1}{2} ft} = 8 \frac{lb}{ft}$

The ODE is

$$\frac{1}{8} x'' + 8x = 0$$

$$\Rightarrow x'' + 64x = 0$$

We have initial conditions

$$x(0) = 4$$

$$x'(0) = -24$$

4 ft
above
equilibrium →

24 ft/sec
downward →

The characteristic equation (in r) is

$$r^2 + 64 = 0$$

$$r^2 = -64$$

$$r = \pm i8 = 0 \pm i8$$

$$x_1 = e^{0t} \cos(8t) \quad , \quad x_2 = e^{0t} \sin(8t)$$

$$x(t) = c_1 \cos(8t) + c_2 \sin(8t)$$

Apply $x(0) = 4 \quad , \quad x'(0) = -24$

$$X'(t) = -8C_1 \sin(8t) + 8C_2 \cos(8t)$$

$$X(0) = C_1 \cos(0) + C_2 \sin(0) = 4$$

1 0 $C_1 = 4$

$$X'(0) = -8C_1 \sin(0) + 8C_2 \cos(0) = -24$$

$$8C_2 = -24 \Rightarrow C_2 = -3$$

The equation of motion is

$$x = 4 \cos(8t) - 3 \sin(8t)$$

The period $T = \frac{2\pi}{\omega} = \frac{2\pi}{8} = \frac{\pi}{4}$

The frequency $f = \frac{1}{T} = \frac{4}{\pi}$

The amplitude

$$A = \sqrt{4^2 + (-3)^2} = 5$$

The phase shift ϕ satisfies

$$\sin \phi = \frac{4}{5} \text{ and } \cos \phi = \frac{-3}{5}$$

$$\phi = \cos^{-1}\left(\frac{-3}{5}\right) \approx 2.21$$

about 127°

Free Damped Motion

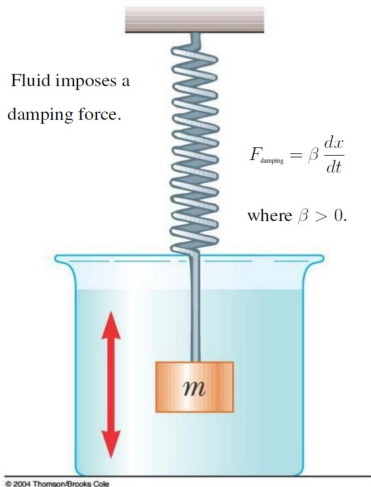


Figure: If a damping force is added, we'll assume that this force is proportional to the instantaneous velocity.

Free Damped Motion

Now we wish to consider an added force corresponding to damping—friction, a dashpot, air resistance.

Total Force = Force of spring + Force of damping

$$m \frac{d^2 x}{dt^2} = -\beta \frac{dx}{dt} - kx \quad \implies \quad \frac{d^2 x}{dt^2} + 2\lambda \frac{dx}{dt} + \omega^2 x = 0$$

where

$$2\lambda = \frac{\beta}{m} \quad \text{and} \quad \omega = \sqrt{\frac{k}{m}}.$$

Three qualitatively different solutions can occur depending on the nature of the roots of the characteristic equation

$$r^2 + 2\lambda r + \omega^2 = 0 \quad \text{with roots} \quad r_{1,2} = -\lambda \pm \sqrt{\lambda^2 - \omega^2}.$$

Case 1: $\lambda^2 > \omega^2$ Overdamped

$$x(t) = e^{-\lambda t} \left(c_1 e^{t\sqrt{\lambda^2 - \omega^2}} + c_2 e^{-t\sqrt{\lambda^2 - \omega^2}} \right)$$

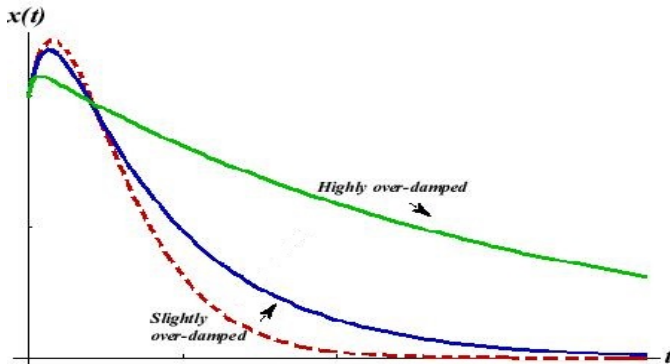


Figure: Two distinct real roots. No oscillations. Approach to equilibrium may be slow.

Case 2: $\lambda^2 = \omega^2$ Critically Damped

$$x(t) = e^{-\lambda t} (c_1 + c_2 t)$$

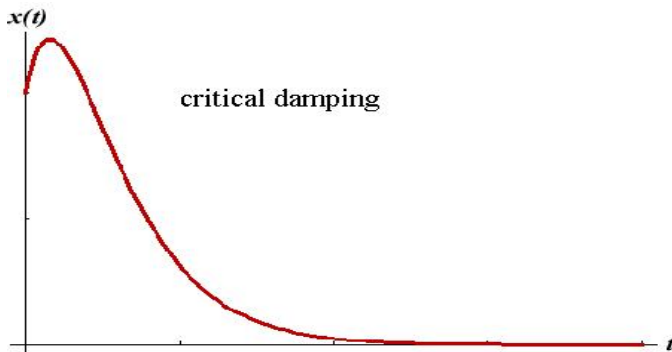


Figure: One real root. No oscillations. Fastest approach to equilibrium.

Case 3: $\lambda^2 < \omega^2$ Underdamped

$$x(t) = e^{-\lambda t} (c_1 \cos(\omega_1 t) + c_2 \sin(\omega_1 t)), \quad \omega_1 = \sqrt{\omega^2 - \lambda^2}$$

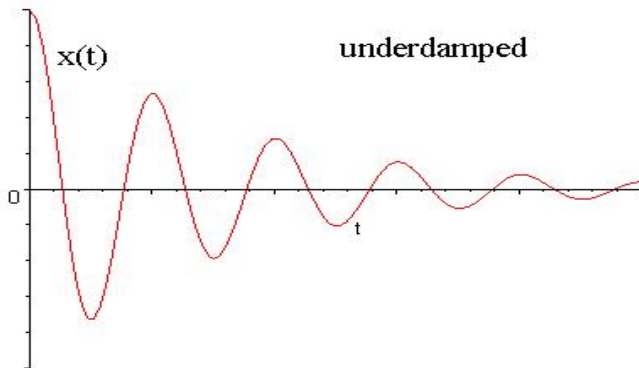


Figure: **Complex conjugate roots.** Oscillations occur as the system approaches (resting) equilibrium.

Comparison of Damping

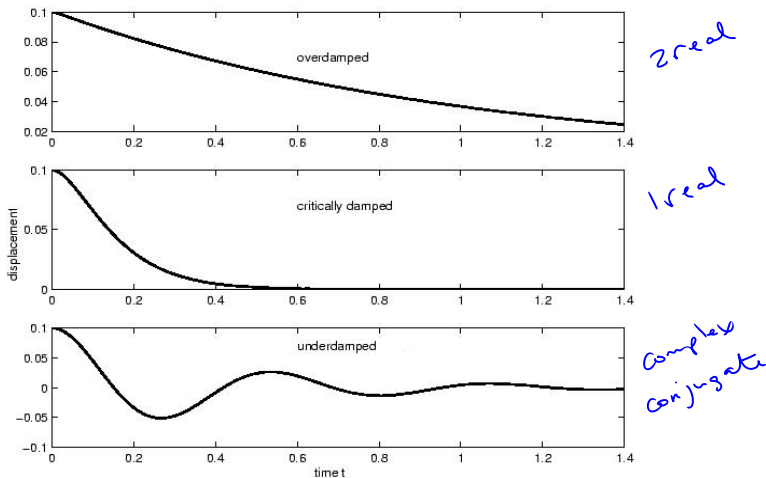


Figure: Comparison of motion for the three damping types.